

A Coupled Discrete/Continuous Method for Computing Lattices. Application to a Masonry-Like Structure

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Abstract

This paper presents a coupled Discrete/Continuous method for computing lattices and its application to a masonry-like structure. This method was proposed and validated in the case of a one dimensionnal (1D) railway track example presented in (Hammoud et al. (2009)). We study here a 2D model which consists of a regular lattice of square rigid grains interacting by their elastic interfaces. Two models have been developed, a discrete one and a continuous one. In the discrete model, the grains which form the lattice are considered as rigid bodies connected by elastic interfaces (elastic thin joints). In other words, the lattice is seen as a “skeleton” in which the interactions between the rigid grains are represented by forces and moments which depend on their relative displacements and rotations. The continuous model is based on the homogenization of the discrete model (Cecchi & Sab (2009)). Considering the case of singularities within the lattice (a crack for example), we develop a coupled model which uses the discrete model in

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singular zones (zones where the discrete model cannot be homogenized), and the continuous model elsewhere. A criterion of coupling is developed and applied at the interface between the discrete and the continuum zones. It verifies the convergence of the coupled solution to the discrete one and limits the size of the discrete zone. A good agreement between the full discrete model and the coupled one is obtained. By using the coupled model, an important reduction in the number of degrees of freedom and in the computation time compared to that needed for the discrete approach, is observed.

Key words: Discrete, Finite Element, Homogenization, Masonry, Interface, Coupling.

1. Introduction

The aim of this paper is to present an application to 2D masonry pannels of the coupled method between discrete and continuum media already proposed and validated in the case of a 1D structure by (Hammoud et al. (2009)). The 1D model consisted of a beam resting on an elastic springs. The deflection of the beam (as well as the nodal parameters) was calculated by using two approaches; a discrete approach and a macroscopic approach deduced from the discrete one. A comparison between the response of the system obtained by using these approaches showed the cases where the macroscopic approach cannot replace the discrete one. This difference leaded us to apply a Discrete/Continuum coupling method. A criterion of coupling was developped. In the coupled approach, the macroscopic scale was the intial scale computation. A local discrete computation was done on each macroscopic element. A comparion was done between the nodal parameters computed by the local discrete method and the continuum one. If a strong difference was observed, a refinement of the computation scale was done. This procedure of refinement was necessary in the zone of singularities.

In this present research, a 2D model will be considered. A masonry pannel can be described by a discrete model or a continuous model. See Alpa and Monetto (1994), Sab (1996), Cecchi & Sab (2002a), Cecchi & Sab (2002b), Cecchi & Sab (2004), Cluni & Gusella (2004), for example. In the discrete model, the blocks which form the masonry wall are modeled as rigid bodies connected by elastic interfaces. Then, the masonry is seen as “skeleton” in which the interactions between the rigid blocks are represented by forces

and moments which depend on their relative displacements and rotations. The second model is a continuous one based on the homogenization of the discrete model.

Many coupled model between the discrete and continuous media are developed. See among others the works of Broughton et al. (1999), Curtin et al. (2003), Wagner et al. (2003), Fish et al. (2004), Xiao & Belytschko (2004), Ricci et al (2005), Frangin et al. (2006) and Klein et al. (2006). In these works, the domain is decomposed into a discrete zone, a continuous zone and an interface zone between the discrete and continuous zones. The interface zone can be a bridging or a handshaking zone where the two descriptions of material exist. Thus, the problem of how to partition energy within the overlap zone is important. For the sake of brevity of the text, the litterature review of these coupled models has been omitted. An exhaustive litterature review has been given in Hammoud et al. (2009). In our model, the handshaking zone is replaced by an interface and then the DoFs of the discrete zone are linked to the DoFs of the continuous zone by calculating the interaction rigidity matrix. The total energy of the domain is written as follows :

$$\mathbf{E}^{\text{total}} = \mathbf{E}^{\text{D}} + \mathbf{E}^{\text{C}} + \mathbf{E}^{\text{C-D}} \quad (1)$$

where \mathbf{E}^{D} and \mathbf{E}^{C} are the elastic energies of the discrete and the continuum zones, respectively. $\mathbf{E}^{\text{C-D}}$ is the energy of the interaction between the discrete element (DE) and the finite element (FE) of these zones.

As for the 1D model (Hammoud et al. (2009)) , the mechanical parameters of the system being studied will be calculated in a way that does not

require the calculation of the energy and avoids the problem of how to partition this energy between the discrete and continuum zones. We will calculate the global rigidity matrices (discrete (\mathbf{K}^D), continuous (\mathbf{K}^C) and interaction (\mathbf{K}^{C-D})) and then solve a linear system written as follows :

$$\underbrace{\left[\left(\begin{array}{c|c} \mathbf{K}^C & 0 \\ \hline 0 & \mathbf{K}^D \end{array} \right) + (\mathbf{K}^{C-D}) \right]}_{\mathbf{K}^{\text{total}}} \begin{pmatrix} \mathbf{U}^C \\ \mathbf{U}^D \end{pmatrix} = \mathbf{F}^{\text{total}} \quad (2)$$

In this present research, at first, we present the 2D masonry model. Secondly, we develop the discrete and the continuous models used to calculate the behavior of the masonry pannel. The continuous boundary value problem is solved by using the Finite Element Method. We implement the full continuous and the full discrete models in a MATLAB code as well as the coupled discrete/continuous one. This case is validated in comparison with a FE software (ABAQUS). We also develop a numerical bench test in order to prove that the discrete medium is homogenizable in the case of no singularities. In the case where singularities exist in the structure (a crack for example), a criterion of coupling between discrete and continuous models, is developed. Near the crack, a discrete zone is used and farther a FE mesh is employed. The criterion of coupling applied at the interface of these zones, verify the convergence of the coupled solution to that discrete. The size of the discrete zone is limited and a considerable reduction of the DoFs is also observed.

2. The discrete model

The 2D model consists of a regular lattice of square rigid grains interacting by their elastic interfaces (see figure 1).

Figure 1 is approximately here

The in-plane motion of the grain can be described by two displacements and one rotation at the center.

The geometry of the lattice is described hereafter. The position of the center of grain $B^{i,j}$, $\mathbf{y}^{i,j}$, in the Euclidean space is formulated as follows :

$$\mathbf{y}^{i,j} = ia\mathbf{e}_1 + ja\mathbf{e}_2 \quad (3)$$

$\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3$ is an orthonormal base.

So the displacement of the $B^{i,j}$ grain is an in plane rigid body motion :

$$\mathbf{u}(\mathbf{y}) = \mathbf{u}^{i,j} + \boldsymbol{\omega}^{i,j} \times (\mathbf{y} - \mathbf{y}^{i,j}), \quad \forall \mathbf{y} \in B^{i,j} \quad (4)$$

where

$$\mathbf{u}^{i,j} = u_1^{i,j}\mathbf{e}_1 + u_2^{i,j}\mathbf{e}_2 \quad \text{and} \quad \boldsymbol{\omega}^{i,j} = \omega_3^{i,j}\mathbf{e}_3 \quad (5)$$

If the mortar joint is modeled as an elastic interface, then the constitutive law is a linear relation between the tractions on the block surfaces and the jump of the displacement :

$$\mathbf{t} = \boldsymbol{\sigma} \mathbf{n} = \mathbf{K} \cdot \mathbf{d} \quad \text{on } S \quad (6)$$

Here, $\boldsymbol{\sigma}$ is the stress tensor, \mathbf{n} is the normal to the interface S and \mathbf{d} is the displacement jump at S . For isotropic mortar, the elastic interface stiffness tensor \mathbf{K} is given as:

$$\mathbf{K} = \frac{1}{e} (\mu^M \mathbf{I} + (\lambda^M + \mu^M)(\mathbf{n} \otimes \mathbf{n})) \quad (7)$$

where λ^M and μ^M are the Lamé constants of the mortar and e is the thickness of the real joint.

The elastic strain energy associated to the interface S is :

$$\mathcal{W} = \frac{1}{2} \int_S \mathbf{d} \cdot (\mathbf{K} \cdot \mathbf{d}) dS \quad (8)$$

Note that each grain has four neighbours that mean four interfaces in which two are horizontal and two are vertical, as shown in figure 2. The vectors $\mathbf{C}^+ \mathbf{M}_1$ and $\mathbf{C}^- \mathbf{M}_1$ are given by :

$$\begin{aligned} \mathbf{C}^+ \mathbf{M}_1 &= -\frac{a}{2} \mathbf{e}_1 + y \mathbf{e}_2 \\ \mathbf{C}^- \mathbf{M}_1 &= \frac{a}{2} \mathbf{e}_1 + y \mathbf{e}_2 \end{aligned} \quad (9)$$

Figure 2 is approximately here

So the displacement of a point located on the vertical interface is written as follows :

$$\begin{aligned} \mathbf{u}^+(M_1) &= \mathbf{u}(C^+) + \boldsymbol{\omega}^+ \times \mathbf{C}^+ \mathbf{M}_1 \\ \mathbf{u}^-(M_1) &= \mathbf{u}(C^-) + \boldsymbol{\omega}^- \times \mathbf{C}^- \mathbf{M}_1 \end{aligned} \quad (10)$$

Thus, the displacement jump at S can be written as :

$$\begin{aligned} \mathbf{d} &= \mathbf{u}^+(M_1) - \mathbf{u}^-(M_1) = d_1 \mathbf{e}_1 + d_2 \mathbf{e}_2 \\ &= (u^+ - u^- + (\omega^- - \omega^+) y) \mathbf{e}_1 + \left(v^+ - v^- - (\omega^- + \omega^+) \frac{a}{2} \right) \mathbf{e}_2 \end{aligned} \quad (11)$$

Let U be the vector of displacement and rotation of two neighbouring grains : $U = [u^+ \ v^+ \ \omega^+ \ u^- \ v^- \ \omega^-]^T$. Then, the elastic strain energy associated to the vertical interface takes the following form :

$$\mathcal{W} = \frac{1}{2} U^T \mathcal{K}_{\text{vertical}} U \quad (12)$$

By using the relationship (8), the value of the elastic strain energy is calculated. So, from (12), we extract the form of the vertical stiffness tensor as follows :

$$\begin{pmatrix} \frac{K'a}{e^v} & 0 & 0 & -\frac{K'a}{e^v} & 0 & 0 \\ 0 & \frac{K''a}{e^v} & -\frac{K''a\sqrt{2}}{4e^v} & 0 & -\frac{K''a}{e^v} & -\frac{K''a\sqrt{2}}{4e^v} \\ 0 & -\frac{K''a\sqrt{2}}{4e^v} & \frac{(K' + 3K'')a}{24e^v} & 0 & \frac{K''a\sqrt{2}}{4e^v} & \frac{(-K' + 3K'')a}{24e^v} \\ -\frac{K'a}{e^v} & 0 & 0 & \frac{K'a}{e^v} & 0 & 0 \\ 0 & -\frac{K''a}{e^v} & \frac{K''a\sqrt{2}}{4e^v} & 0 & \frac{K''a}{e^v} & \frac{K''a\sqrt{2}}{4e^v} \\ 0 & -\frac{K''a\sqrt{2}}{4e^v} & \frac{(-K' + 3K'')a}{24e^v} & 0 & \frac{K''a\sqrt{2}}{4e^v} & \frac{(K' + 3K'')a}{24e^v} \end{pmatrix} \quad (13)$$

Similarly, the form of the horizontal stiffness tensor $\mathcal{K}_{\text{horizontal}}$ can be found. Hence, the vector of all in-plane degrees of freedom of the structure is calculated by solving the following linear system :

$$\mathbb{K}\mathbb{U} = \mathbb{F} \quad (14)$$

in which $\mathbb{U} = [u_1 \ v_1 \ \omega_1 \ \dots \ u_N \ v_N \ \omega_N]^T$ is the vector of all in-plane degrees of freedom of the structure under consideration and

$\mathbb{F} = [f_1 \ t_1 \ m_1 \ \dots \ f_N \ t_N \ m_N]^T$ is the vector of all in-plane elastic actions. \mathbb{K} is the in-plane stiffness matrix calculated by assembling the vertical and horizontal interfaces of the structure.

3. The continuum model

The homogenization of periodic discrete materials has been previously presented in (Pradel & Sab (1998a), Pradel & Sab (1998b), Florence & Sab (2005) and Florence & Sab (2006)), for example. The geometry will be discretized by using the Finite Element Method. As mentioned above, the implementation of the homogenized model will be done with a Matlab code in order to couple later, a continuum zone to a discrete one.

Let us consider the static case of the elastic behavior of the domain. The equilibrium equation is written as :

$$\nabla \boldsymbol{\sigma} + \mathbf{b} = 0 \quad (15)$$

where ∇ is the divergence operator, $\boldsymbol{\sigma}$ is the Cauchy stress tensor and \mathbf{b} the external load applied on the domain. The stress-strain relationship is given by :

$$\boldsymbol{\sigma} = \mathbb{C} : \boldsymbol{\epsilon} \quad (16)$$

Where \mathbb{C} is the homogenized elastic tensor and $\boldsymbol{\epsilon}$ is the strain tensor.

Using a weak variational formulation, the equilibrium equation (15) is written as follows :

$$\mathbb{K} \mathbf{U} = \mathbb{F} \quad (17)$$

where \mathbb{K} is the global stiffness matrix of the domain, \mathbf{U} is the global vector of nodal displacements and \mathbb{F} is the global vector of external forces applied on the finite element nodes.

In other words, \mathbb{K} and \mathbb{F} are the assembling of the elementary matrix \mathbb{K}^e and the force vector \mathbb{F}^e , respectively.

\mathbb{C} is the homogenized elastic tensor. It is written as follows :

$$\mathbb{C} = \begin{bmatrix} A_{1111}^{\text{hom}} & 0 & 0 \\ 0 & A_{2222}^{\text{hom}} & 0 \\ 0 & 0 & A_{1212}^{\text{hom}} \end{bmatrix} \quad (18)$$

where $A_{1111}^{\text{hom}} = \frac{K'a}{e^h}$, $A_{2222}^{\text{hom}} = \frac{K'a}{e^h}$ and $A_{1212}^{\text{hom}} = \frac{2K''a}{e^h}$.

$K' = \lambda^M + 2\mu^M$, $K'' = \mu^M$ and e^h is the thickness of the joint between two grains.

4. Numerical simulations

4.1. Discrete model versus continuous model

4.1.1. Compression test

Let us consider a panel (width L and height H) subjected to compression actions, supported at its left and right edges with $u_2(X=0, X=L) = 0$, fixed at the base $u_1(Y=0) = u_3(Y=0) = \omega_3 = 0$ and loaded with a vertical uniform force applied on the upper edge (see figure 3). In this test, any heterogeneity is introduced in the panel.

Figure 3 is approximately here

In the discrete model, the uniform load is applied on each grain center of the upper edge. In the continuous one, the load is applied at the nodes of the finite element. In figure 4, the nodal displacements of the middle line of the panel ($Y = \frac{H}{2}$), u_2 , are represented. We observe a good match between the discrete and continuous displacements. This matching means that the discrete medium is homogeneizable and the continuous model can replace correctly the discrete one.

Figure 4 is approximately here

4.1.2. Shear test

The case of shear stress is investigated in this part. In the discrete model, the panel is under the following boundary conditions:

grains at the top of the panel: uniform horizontal force,

grains in the left side of the panel $u_2 = 0$ and u_1, ω_3 are free,

grains in the right side of the panel $u_2 = 0$ and u_1, ω_3 are free,

blocks at the base of the panel $u_1 = u_2 = 0$ and ω_3 is free.

In the continuous model, the boundary conditions are the following:

$u_2(x_1 = 0) = u_2(x_1 = L) = 0$, $u_1(x_1 = 0) = u_1(x_1 = L) = 0$ and a horizontal uniform load is applied at the side $x_2 = H$ (see figure 5). As in the compression test, the medium is considered homogeneous.

Figure 5 is approximately here

By considering the discrete medium at coarse scale and the continuous model at fine scale, it is obtained that the u_1 displacements of the middle line of the panel don't match correctly and the relative difference is more than 10% (figure 6). If we refine the coarse scale of the discrete medium, this difference will be negligible as we can observe on the (figure 6).

Figure 6 is approximately here

Finally, we conclude that the discrete solution converges to the continuous one when the computation scale is fine. This convergence also means, that the discrete medium is homogeneizable and the continuous model can

replace correctly the discrete one when there is no singularities in the structure.

It is clear that the computation time and the number of degrees of freedom (DOFs) in the discrete model are more important than that of the continuous model. In what follows, in the case of the shear test studied above, a simple comparison (table 1) shows the importance of these two factors: computation time and gain in DOFs.

Table 1 is approximately here

4.2. The coupling model

Now we consider a crack in the panel. Near this crack the medium cannot be homogenized. It is noted that the discrete model can be used to simulate all the medium, but taking into account the computation time and the number of DOFs, it will be better if we can couple the continuous and discrete models, then the discrete model is used in the cracked zone and the continuous one is used elsewhere.

4.2.1. Principle of the coupling model

The medium is decomposed into two regions. The first one is the continuum region modeled by finite elements (rectangular with two DoFs by node), the second is the discrete region where the Discrete Element (DE) are the centre of grains (3 DoFs at the center of grains). At the interface between these zones, interpolated DE are used to link the FE of the continuum zone to the DE of the discrete zone (see figure 7).

Figure 7 is approximately here

As mentionned before, by noting \mathbf{E}^D the elastic energy of the discrete zone, \mathbf{E}^C the elastic energy of the continuum one and \mathbf{E}^{C-D} the energy of the interaction between the DE and the FE, the total energy of the coupled medium is given by (1):

The interaction energy between two DEs (- and +) is written as follows :

$$\mathbf{E}^I = \frac{1}{2} \begin{pmatrix} \mathbf{U}^- \\ \mathbf{U}^+ \end{pmatrix}^T [\mathbf{K}^I] \begin{pmatrix} \mathbf{U}^- \\ \mathbf{U}^+ \end{pmatrix} \quad (19)$$

\mathbf{E}^I and \mathbf{K}^I are the interaction energy and the stiffness matrix of the interface between two adjacents grains, respectively. \mathbf{U}^- and \mathbf{U}^+ are the vectors of displacements and rotation of the grains (-) and (+), respectively. If we consider a FE modeled by DEs, a relationship between the displacement of the FE node's (\square) and the displacement of the DE (\circ created inside the FE) can be established by interpolation, using the shape functions. By noting $[U, V, W]^T$ the vector of displacements and rotation of a DE and $[u_1, v_1, u_2, v_2, u_3, v_3, u_4, v_4]^T$ the vector of nodal displacements of a FE, the relationship writes :

$$\begin{bmatrix} U, V, W \end{bmatrix}^T = \mathbf{D} \begin{bmatrix} u_1, v_1, u_2, v_2, u_3, v_3, u_4, v_4 \end{bmatrix}^T \quad (20)$$

\mathbf{D} is a interpolation matrix.

It is noted that the discrete displacement at the center of the grain (\mathbf{U}) is equal to the finite displacement interpolated in the center of the grain ($\mathbf{u}(\mathbf{x})$): $\mathbf{U} = \mathbf{u}(\mathbf{x})$. The discrete rotation is also in relation with the finite displacement by: $\mathbf{W} = \frac{1}{2} (\mathbf{grad} \mathbf{u}(\mathbf{x}) - \mathbf{grad}^T \mathbf{u}(\mathbf{x}))$ in which \mathbf{x} is the vector position of the grain center's.

At the same time, each DE located at the edge of the discrete zone (B^D) is connected to an interpolated DE located at the edge of the continuum zone (B^C) by adding half of the interaction energy (19) to the total elastic energy.

Thus, from these two relationships, a DE located in the discrete zone is linked to a FE in the continuum zone. If we use (20) for the interpolated DE (\mathbf{U}^- or \mathbf{U}^+), then the interaction energy(19) between the DE and the FE will be a quadratic function of \mathbf{U}^D and \mathbf{U}^C .

\mathbf{U}^D and \mathbf{U}^C are the global displacements vector of the discrete and continuum zones respectively. By designing \mathbf{K}^D and \mathbf{K}^C , the discrete and continuum stiffness matrices, the total energy of the medium will be :

$$\mathbf{E}^{\text{total}} = \frac{1}{2} \begin{pmatrix} \mathbf{U}^C \\ \mathbf{U}^D \end{pmatrix}^T \underbrace{\left[\begin{pmatrix} \mathbf{K}^C & 0 \\ 0 & \mathbf{K}^D \end{pmatrix} + (\mathbf{K}^{C-D}) \right]}_{\mathbf{K}^{\text{total}}} \begin{pmatrix} \mathbf{U}^C \\ \mathbf{U}^D \end{pmatrix} \quad (21)$$

\mathbf{K}^{C-D} is the global matrix of interaction which is calculated by the summation of all elementary interaction matrices between the discrete and continuous zones.

4.2.2. Criterion of coupling

Such as for the 1D methodology (see Hammoud et al. (2009)), a criterion of coupling is developed to limit the size of the discrete zone used in the singular zone. The idea is to apply discrete external forces and moments on the DE located at the edge of a FE near the interface zone and to compare the discrete responses of the grains inside the FE to their interpolated FE responses.

Figure 8 is approximately here

The external loading is computed as follows: Using (20), the displacements at the center of the interpolated DE created in the FE can be calculated. From the interaction energy formulated in (19), we calculate the interaction forces and moments between these two DEs using the relation $(\mathbf{F} = [\mathbf{K}^{\text{interface}}] \cdot [\mathbf{U}^+, \mathbf{U}^-]^T)$. All the interaction forces between a DE (\bullet) and an external interpolated DE (\circ) at the edge of FE are computed and assembled to form the external global load applied on the discrete zone included in the FE.

Using the discrete model, we calculate the discrete displacements of the DE noted as \mathbf{U}_a^d . After that, we calculate the difference between the interpolated continuum displacements in (20), \mathbf{U}_i^c at the center of grains and \mathbf{U}_a^d . This difference will be the criterion for coupling. It is formulated as follows :

$$\text{error} = \left| \frac{\mathbf{U}_a^d - \mathbf{U}_i^c}{\mathbf{U}_a^d} \right| \quad (22)$$

By noting “TEST ZONE” the FE zone neighbouring the discrete one, we check the criterion (22) on each FE of this zone. In other words, we check if the FEs of the “TEST ZONE” lead to the correct solution (we mean by it the full discrete solution). So if the error (22) is more than 10%, the scale of computation will be that of the discrete model. The size of the discrete zone will increase. In the other case (error less than 10%), the continuum scale of computation is adapted and the size of the discrete zone is adequate. Due to this criterion, the size of the discrete region is controlled and the number of DoFs is reduced.

4.2.3. Numerical algorithm

At first, the medium is meshed at a coarse scale by using FE. At the center of the medium, a crack is created by broking the interaction between interfaces. The cracked zone is modeled by DE. The size of the discrete zone is fixed. After applying a traction load, for example, we simulate the response of the medium. At the interface between the discrete zone and the continuum one, we check the criterion of coupling described before. Hereafter, a diagram of this algorithm is presented.

Figure 9 is approximately here

4.2.4. Cracked wall: discrete model vs coupled model

We consider a panel (width L , height H) with a crack at its center. The cracked zone is modeled by DE and the rest of the panel is modeled by FE as shown in figure 10.

Figure 10 is approximately here

Firstly a complete discrete simulation is done in order to compare the coupled solution to that discrete. Let us consider a panel modeled by 25×25 grains. After a traction load, we can observe the crack, by simply representing the position of the center of each grain. We can observe in figure 11 the rotation of grains considered like rigid bodies.

Figure 11 is approximately here

In this discrete simulation, the number of DoFs is 625×3 and the computation time is estimated to 322 seconds.

Now, let us consider the coupled simulation. The size of a FE is supposed equal to 8 times the size of a DE. The size of the discrete zone is fixed to 3×3 FEs which means 72 DEs. The mesh after loading takes the shape seen in figure 12.

Figure 12 is approximately here

If we compare the Y displacements of the middle line of the panel, we can observe a perfect match between the discrete and coupled solutions. This agreement is illustrated in figure 13.

Figure 13 is approximately here

4.2.5. *Gain in time and DoFs*

In this paragraph, we underline the advantage of this coupled approach. In the coupled simulation done before, by considering the same dimensions of the panel ($4 \times 4 m^2$), the total number of DoFs is the sum of (72×3 discrete DoFs) and (85×2 continuous DoFs).

The computation time is estimated to 54 seconds. By a simple comparison (see table 2) between discrete and coupled parameters, we can concluded their importance.

Table 2 is approximately here

The gain in DoFs is evaluated to : $G_{\text{DoFs}} = \frac{1875}{386} = 4.86$ and the gain in computation time is : $G_{\text{time}} = \frac{322}{54} = 5.96$. These gain factors will be more interesting in 3D simulations.

By applying the numerical criterion at the interface between discrete and continuum zones, the difference between discrete and continuum solutions is evaluated to 9%. This difference can be minimised if we increase the size of the discrete zone. Thus, the gain in DoFs and computation time will decrease.

5. Conclusion

In this work a 2D coupled model between discrete and continuum media has been performed. The discrete model is based on interaction between rigid bodies by their interfaces. The continuous model is based on the homogenization of the discrete model. Numerical simulations show that the discrete medium is homogeneizable if there is no singularities in the medium. Thus, the continuous model can replace correctly the discrete one. When the medium represents some singularities, a coupled model will be developed. The discrete zone is used to simulate the singularities and elsewhere the continuum zone is used. At the coupling interface, a criterion of coupling is developed. With this criterion, we check if the FEs of the interface leads to the full discrete solution. Another contribution of this coupled method is the sensible gain in terms of DoFs and computation time. The results show a perfect match between the full discrete and coupled discrete/continuous solutions. Thus, it would be more interesting to see the impact of this method on a large structure in 3D simulations. In future works, a code with the ability of remeshing many singularities can be generated. We can study the propagation of many cracks considered in discrete zones. It is also interesting to study the dynamic case and the

possiblity of spurious reflections at the interface of coupling.

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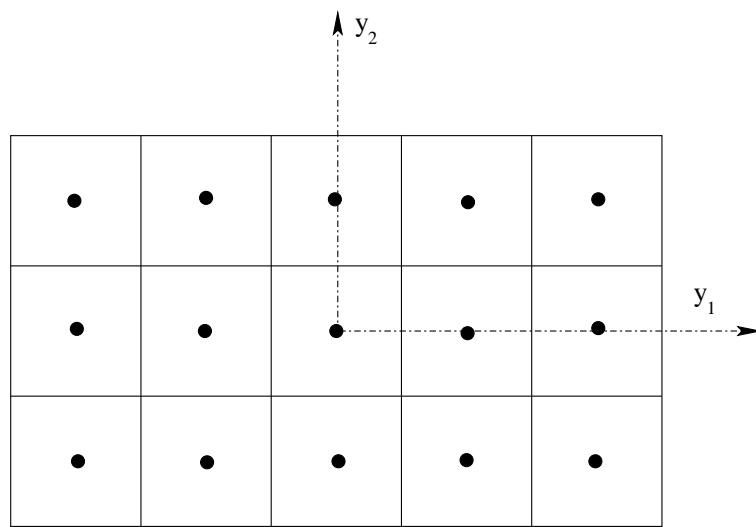


Figure 1: Square grains forming the regular lattice.

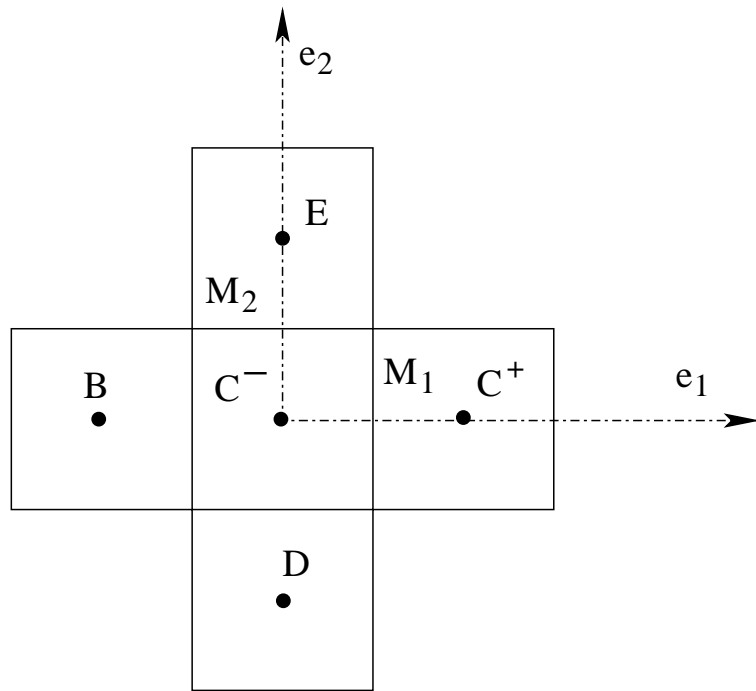


Figure 2: Horizontal and vertical interfaces of a grain

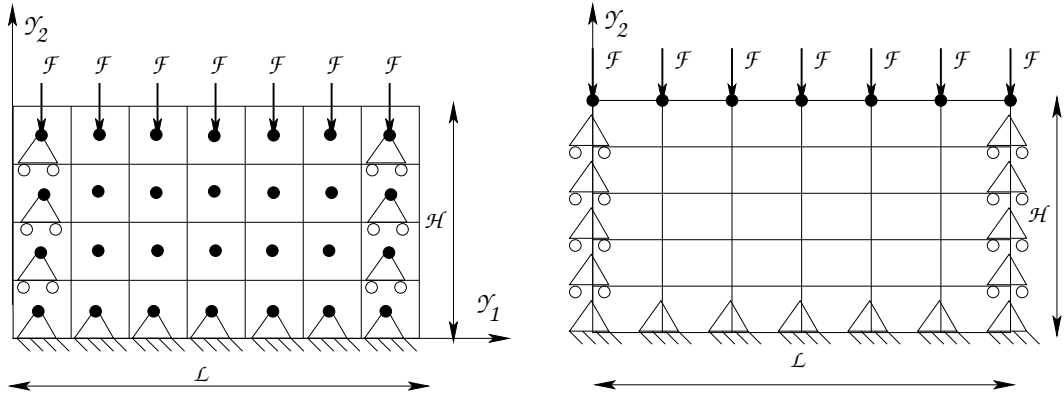


Figure 3: Masonry panel (width L and height H) subject to compression actions supported at its left and right edges $u_2 = 0$, fixed at the base loaded with a vertical uniform force applied on the upper edge: (a) discrete model, (b) continuous model.

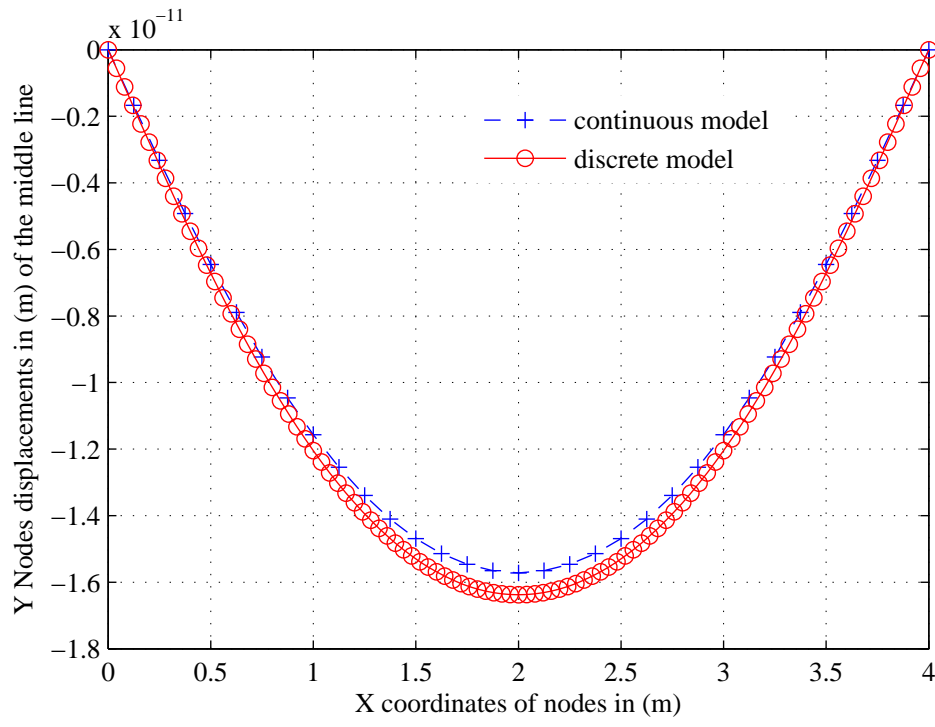


Figure 4: Comparison between discrete and continuous displacements of the nodal line ($Y=H/2$); compression test

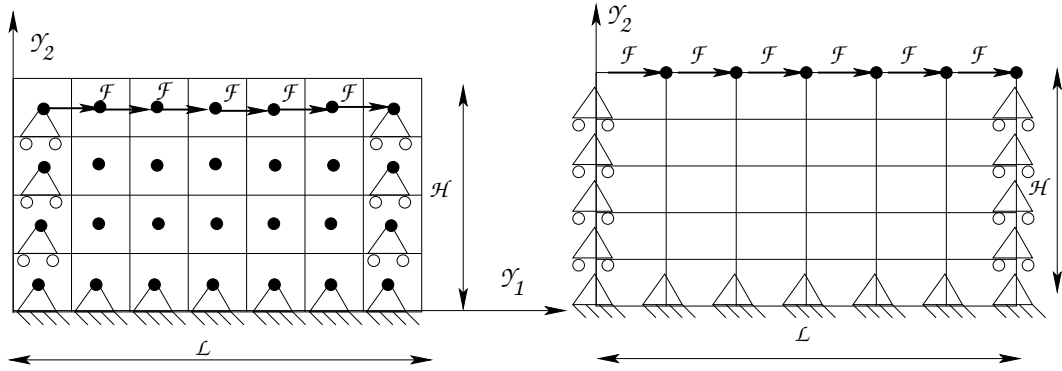


Figure 5: Masonry pannel (width L and height H) subject to shear actions simply supported at its left and right edge $u_2 = 0$ and fixed at the base loaded with a horizontal uniform force applied on the top : (a) discrete model, (b) continuous model.

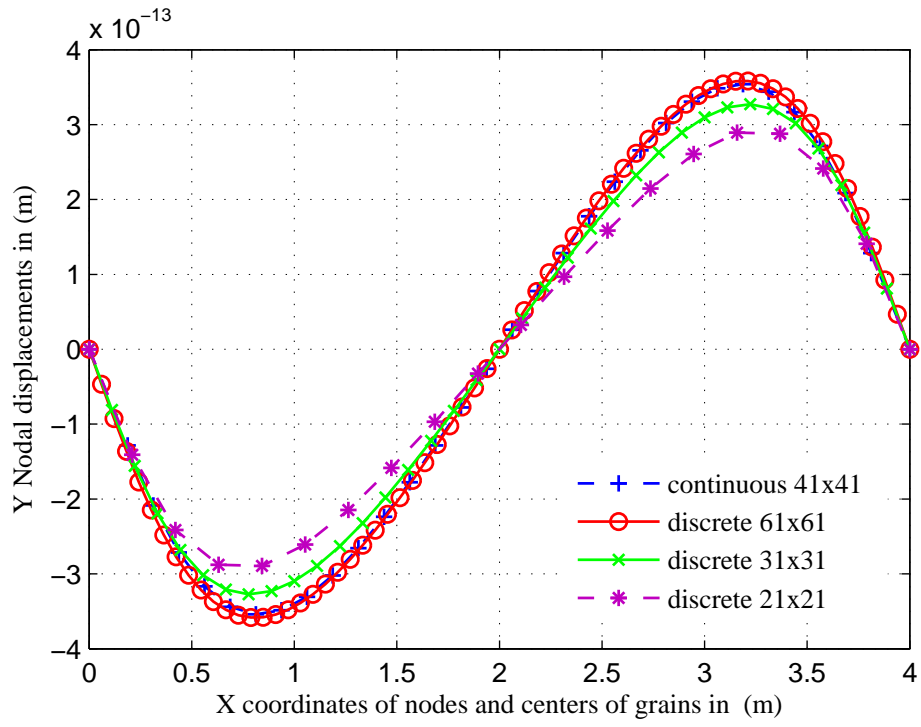


Figure 6: Comparison between discrete and continuous displacements of the middle line ($Y=H/2$); shear test

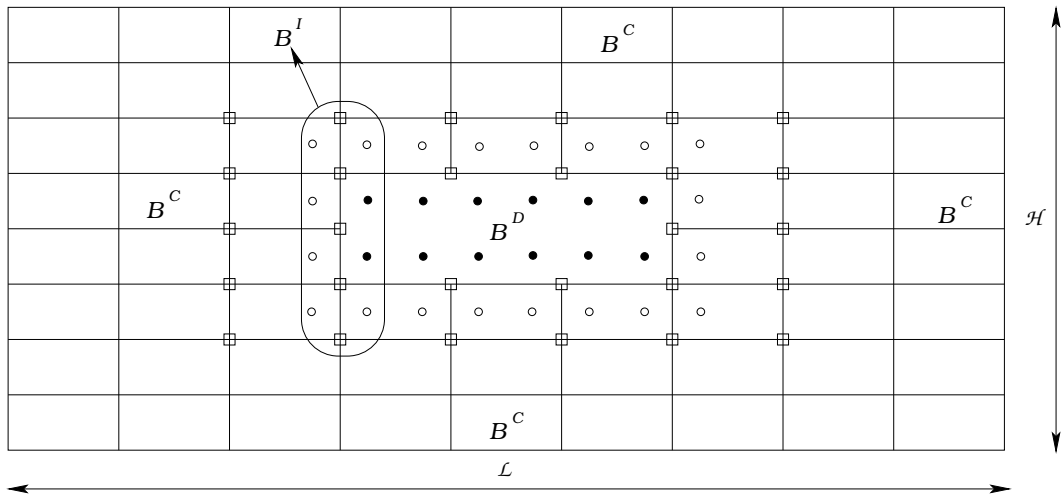


Figure 7: Regular lattice of square grains modeled by a coupling discrete/continuum model; (\bullet) are the DE of the region (B^D) , (\circ) are the interpolated DE of the (B^I) and (\square) are the finite element nodes of the region (B^C)

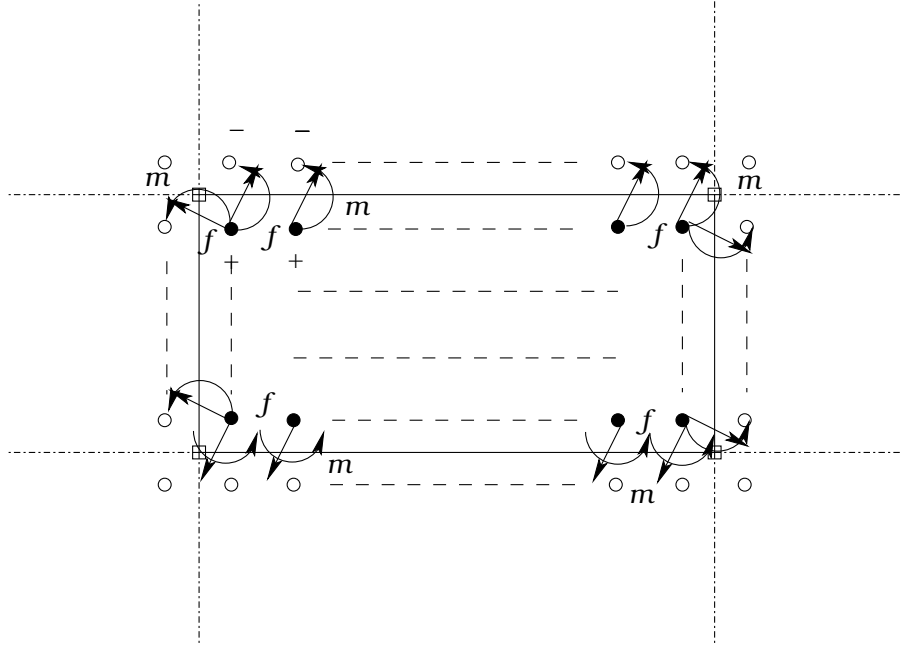


Figure 8: (f, m) are the forces and the moments of interaction between DEs inside the considered FE (\bullet) and interpolated DEs (\circ) inside adjacent FEs.

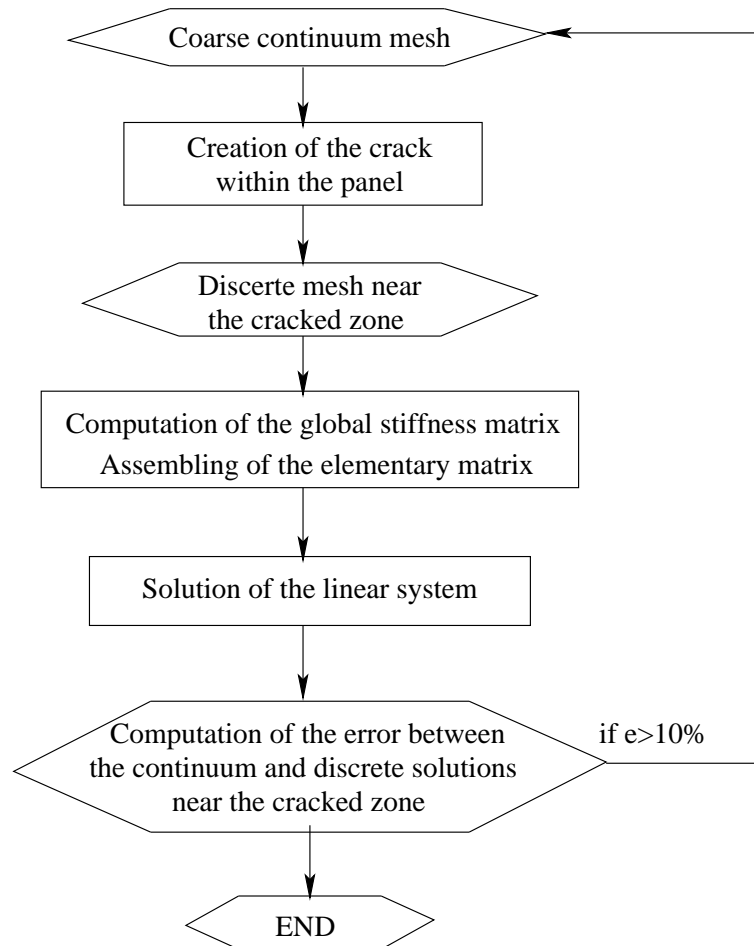


Figure 9: Numerical algorithm of the coupling model

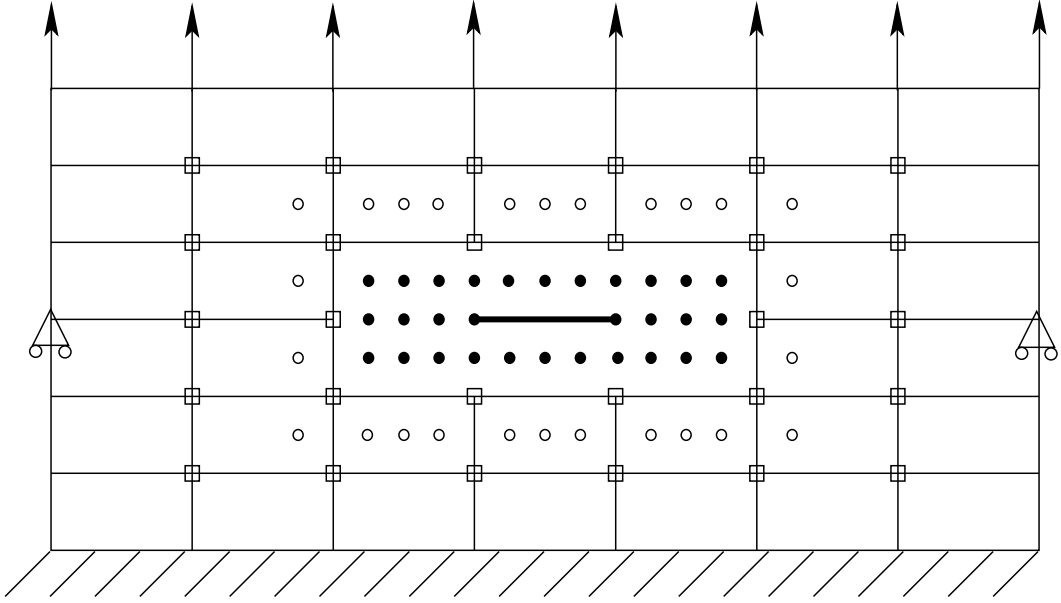


Figure 10: Masonry panel (width L and height H) subject to traction actions, fixed at the base and simply supported at its left and right edges $u_1(Y = H/2, X = 0) = u_1(Y = H/2, X = L)$, loaded with a vertical uniform force applied on the bottom

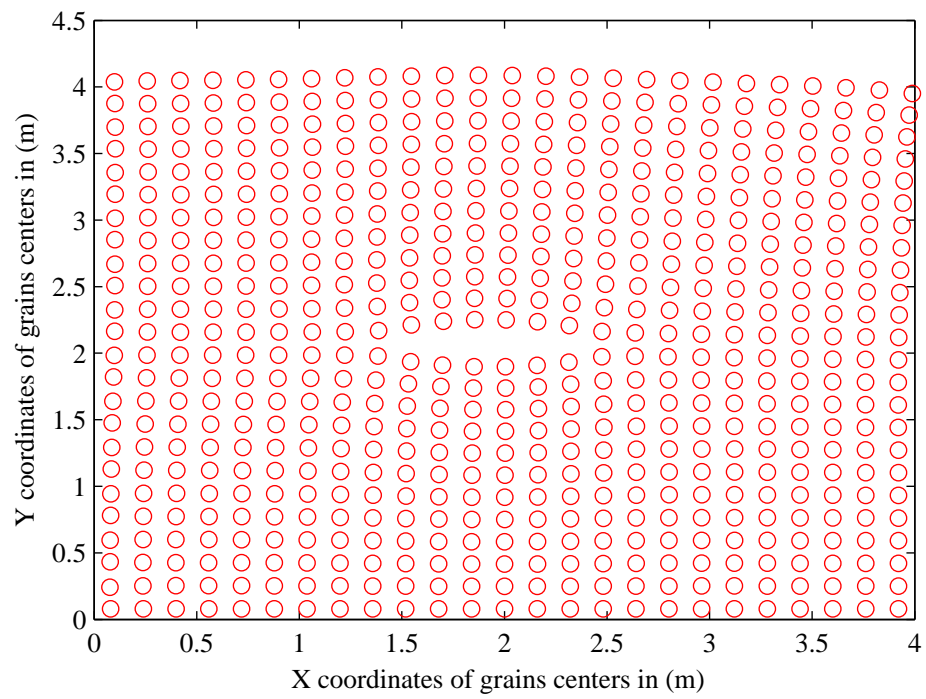


Figure 11: Discrete simulation of the crack in the panel

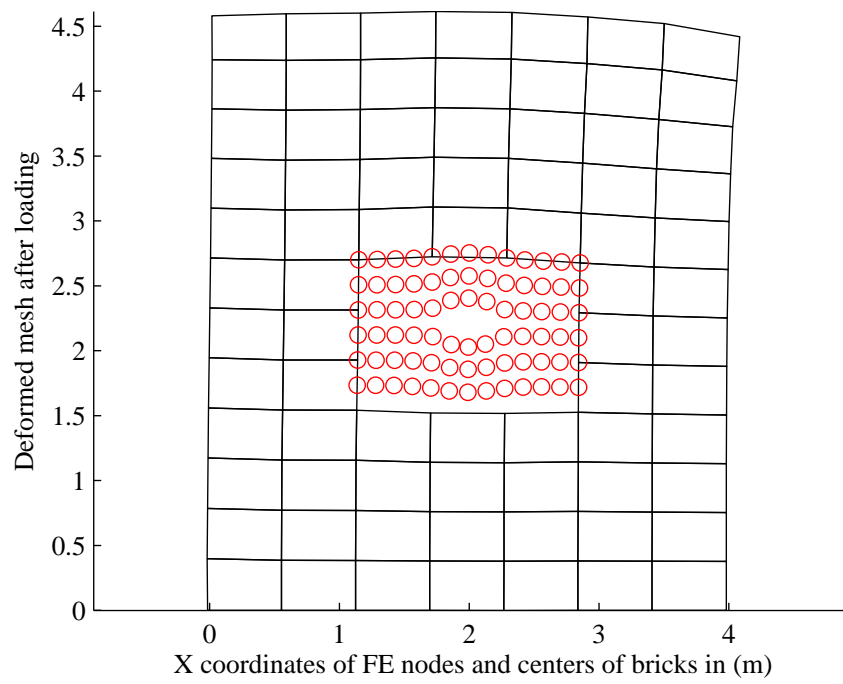


Figure 12: Coupled simulation of the crack in the panel

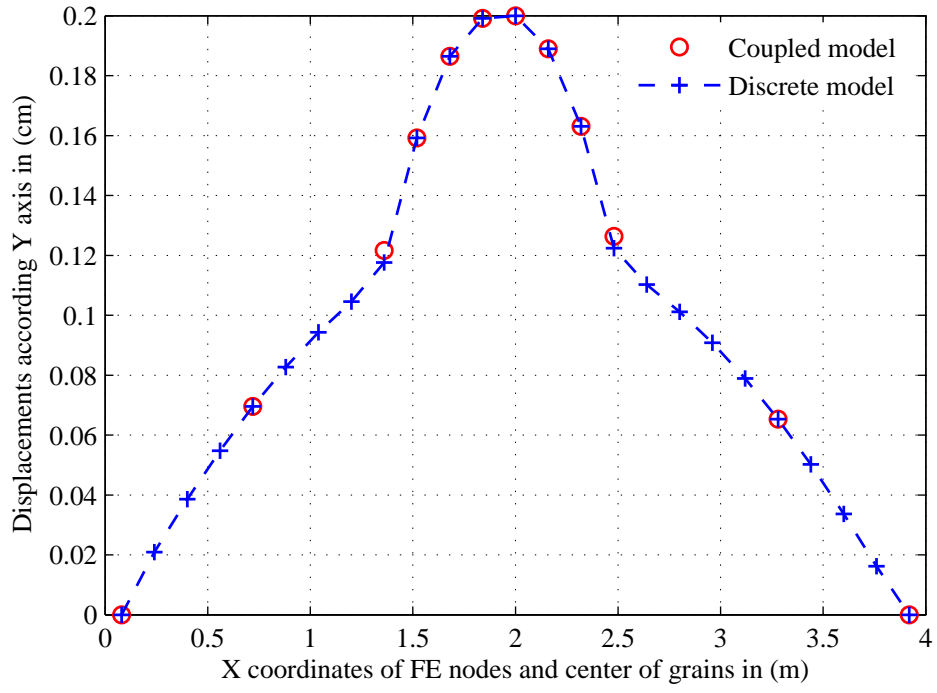


Figure 13: Comparison between discrete and coupled displacements of the middle line ($Y=H/2$); traction test

	Number of nodes	Number of DoFs	Computation time
Discrete model	625	$625 \times 3 = \mathbf{1875}$	322 seconds
Continuous model	72	$72 \times 2 = \mathbf{144}$	35 seconds

Table 1: Gain in computation time and in DoFs; discrete and continuous models

	Number of nodes	Number of DoFs	Computation time
Discrete model	625 DEs	$625 \times 3 = \mathbf{1875}$	322 seconds
Coupled model	85 FEs + 72 DEs	$85 \times 2 + 72 \times 3 = \mathbf{386}$	54 seconds

Table 2: Gain in computation time and in DoFs; coupled and discrete models